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Crush responses of composite cylinder under quasi-static and dynamic loading

Louis N. S. Chiu^{1,2}, Brian G Falzon³, Dong Ruan⁴, Shanqing Xu⁴, Rodney S Thomson^{2,5}, Bernard Chen¹, Wenyi Yan^{1*}

1, Department of Mechanical and Aerospace Engineering, Monash University, Clayton Campus, Victoria 3800, Australia

2, Cooperative Research Centre for Advanced Composite Structures, 1/320 Lorimer Street, Port Melbourne, Victoria 3207, Australia

3, School of Mechanical and Aerospace Engineering, Queen's University Belfast, Belfast, BT9 5AH, UK

4, Faculty of Science, Engineering and Technology, Swinburne University of Technology, John Street, Hawthorn, VIC 3122, Australia

5. Advanced Composite Structures Australia, 1/320 Lorimer Street, Port Melbourne, Victoria 3207, Australia

*Corresponding author. Email: wenyi.yan@monash.edu

Abstract

Despite the abundance of studies investigating the performance of composite structures under crush loading, disagreement remains in the literature regarding the effect of increased strain rate on the crush response. This study reports an experimental investigation of the behaviour of a carbon-epoxy composite energy absorber under static and dynamic loading with a strain rate of up to 100 s^{-1} . Good consistency, among samples tested under the same strain rate, was achieved in the identified damage modes and the measured force responses. The energy absorption was found to be independent of strain rate as the total energy absorption appeared to be dominated by fibre-dominated fracture, which is independent of strain rate within the studied range. The results from this study can inform the design of energy absorbing structures.

Keywords: Composites, Crushing, Strain rate effect, Dynamic, Energy absorption

1. Introduction

Interest in energy absorbing structures for crashworthiness applications has been growing due to the increasingly safety-conscious environment in which the aviation industry operates. Design guidelines [1] and standards [2] impose a maximum allowable acceleration envelope experienced by the occupant in a crash. Composite materials have been gaining popularity in aircraft structures due to their superior specific strength and stiffness, corrosion and fatigue resistance. Their complex failure modes [3-5] facilitate a high level of energy dissipation making them suitable for use in energy absorbing structures to meet crash protection requirements. Jackson et al [6] showed that composite energy absorbing structures can significantly reduce the acceleration experienced by the occupants in a crash environment which would lead to a reduced risk and severity of injury. Hence, the performance of composite structures under crush loading is of great interest.

Performance of energy absorbing structures can be measured through their specific energy absorption (SEA), peak force (F_{peak}), steady-state force (F_{SS}) and crush efficiency (CE). The energy absorption is area under the force (F) – displacement (x) curve, and is directly related to protective capability. SEA is defined as the energy absorption of unit mass (m) of structure (Eq. (1)). Hence, for weight-conscious applications such as aviation, SEA is a critical measure of performance.

$$SEA = \frac{\text{energy absorbed}}{\text{mass of structure consumed}} = \frac{\int_{x_{start}}^{x_{end}} F dx}{m} \quad (1)$$

F_{peak} is the highest force (hence highest acceleration) experienced during the crush event and is directly related to the potential for injury suffered by the occupants in a crash situation. One of the purposes of the energy absorber is to keep acceleration levels within human tolerance limits. F_{SS} is the mean force during steady-state crushing of the specimen after the consumption of the trigger and the passing of peak force, and is a good indicator of the overall energy absorption capability

of the structure. CE [7] (Eq. (2)) is the ratio between F_{SS} and F_{peak} (Eq.(2)) and is indicative of the nature of the crush response.

$$CE = \frac{F_{SS}}{F_{peak}} \quad (2)$$

A catastrophic failure is characterised by a high peak force as well as a low steady-state crushing force, which leads to a reduced crush efficiency.

A wide variety of composite energy absorber configurations have been reported in the literature since the pioneering work by Thornton [8] and Farley [9]. Simple geometries such as circular tubes [9-12], rectangular tubes [13-15] and flat plates [13, 16] have been studied extensively.

Other more complex geometries have also been investigated, including C sections [17, 18] and I sections [15]. Triggering has been shown to be an important aspect of energy absorbent structure design, with chamfering on the loading surface being most common [19]. Other trigger mechanisms studied include the tulip [12, 20] and ply drop [21]. Some authors have designed the structures so that they are self-triggering, for example, using corrugated [22] or hourglass profiles [20]. Others have evaluated different types of material systems and layup configuration [19, 23]. However, most of this experimental data was based on quasi-static testing, despite crushing being a dynamic event. Consequently, the effectiveness of energy absorbers can only be reliably determined once an assessment of possible rate dependence is made.

Currently, there is disagreement in the literature over the effect of intermediate nominal loading rate between 0.1 and 100 s^{-1} on the response of composite structures. Initial tests completed by Thornton [8] suggested rate independence for glass and graphite-epoxy cylindrical tubes. Farley [10] noted that for chamfered cylinders, $[0/\pm \theta]_2$ graphite-epoxy specimens were rate insensitive, but Kevlar-epoxy and $[\pm \theta]_3$ graphite-epoxy specimens displayed increased specific energy absorption (SEA) as the testing speed increased. Palanivelu et al. [7] found the SEA of specimens

with a circular cross-section were rate insensitive, whereas those with a square cross-section increased slightly with crush speed. Work done on rectangular tubes by Mamalis et al. [24] has found that both the SEA and peak force of square tubes increased with respect to increasing strain rate. On the other hand, crush testing conducted by Jackson et al. [17] on chamfered C-section found an approximately 10% reduction in SEA for specimens impacted at 8.5 m/s when compared, with those crushed at 20 mm/s. This was confirmed by David et al. [18] who also observed a reduced SEA on dynamically tested C-section specimens with a $[(0/90)_2/0/(90/0)_2]$ layup. Brighton et al. [25] reported a decrease in SEA for chamfered carbon-epoxy tubes with a $[0/90]_4$ layup when the test speed was increased whereas the rate effect for a chamfered 4 ply glass-polypropylene fabric tubes were inconclusive.

A sufficiently large sample size is often required to measure the scatter in the experimental data. Brighton et al. [25] noted that the lack of manufacturing control has a significant effect on the specimen response which leads to possible strain rate effect being hidden within the noise. Unstable collapse of specimens [24] also presented challenges in their measurement due to the presence of high force spikes in the resulting force response. Furthermore, unstable crushing response is also more dependent on any microscopic defects or weak points within the structure, which are random in nature. In order for a conclusion to be drawn with confidence, the observed trend must be compared with the size of the scatter inherent in the experimental results.

This study presents a comparison between the response and damage mechanisms of a tulip triggered composite cylinder subjected to quasi-static and dynamic crush loading with a strain rates of up to 100 s^{-1} . Qualitative analysis of the specimens was conducted to identify the damage mode and their propagation through the structure. Quantitative analysis of the force response for each test condition was also completed. The reliability of the results was assessed through analysing the scatter of the measured data.

2. Experimental method

2.1. Specimen design

The specimen is a cylindrical tube with a series of tulip triggers cut into the top surface. The tubes were manufactured using a unidirectional Hexcel HexPly T700/M21 carbon-epoxy prepreg with a $[0/90/0/90]_s$ layup. The tubes were autoclave cured as per manufacturer's specifications. The effect of seams is minimised by placing seams of adjacent plies on opposing sides of the cylinder. After curing, the composite tubes were machined to the geometry specified in Figure 1.

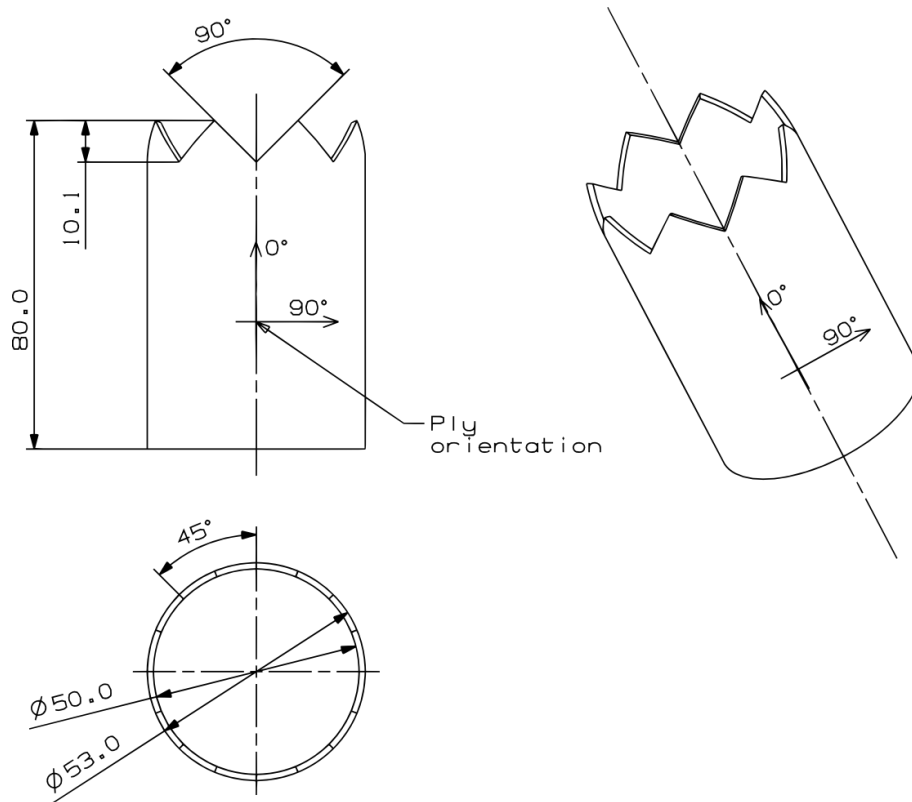


Figure 1: Test specimen: cylindrical tube topped with tulip triggers.

A tubular geometry was selected to facilitate their testing as free-standing energy absorbers. A circular cross-section also avoids stress concentrations, which leads to a more progressive and localised damage progression [19], resulting in a higher energy absorption in comparison with a square or rectangular cross-section. The triangular tulip trigger can achieve a higher steady-state crush load [19] in contrast to hourglass [20], crown and chamfer [26] triggers, leading to increased

energy absorption. The increased length of the trigger region helps to spread the force spike in the initial stage of impact and reduce the peak transmitted force. A balanced $[0/90/0/90]_s$ layup was chosen to ensure lateral confinement and support within the orthogonal plies. Jacob et al. [27] noted increased overall energy absorption when fibres parallel to the loading direction were laterally supported, leading to increased fracture of these fibres. Commercially available filament wound tubes were not considered due to a lack of fibres parallel to the loading direction.

2.2. Test method

MIL-STD-1290A [2] sets out that airframes should be designed for at least an 8 m/s vertical speed at impact. NASA [28] conducted drop tests on representative fuselage sections with attached energy absorbers at impact speeds between 7.3 to 11.6m/s. This corresponded to a nominal strain rate of between 15 and 23 s^{-1} in these absorbers, which represented the likely strain rates experienced in a survivable crash scenario. Hence, the chosen strain rates for this study were 2.1×10^{-4} , 0.2, 30, 60 and 100 s^{-1} to encompass likely strain rates that an energy absorber would be exposed to in the event of a crash. The chosen specimens were tested at a range of different strain rates, as detailed in Table 1, along with the number of specimens associated with each test configuration.

Table 1: Test conditions

	Low loading rate		High loading rate		
Testing machine type	Screw-driven		Hydraulic		
Nominal strain rate	$2.1 \times 10^{-4}\text{ s}^{-1}$	0.2 s^{-1}	30 s^{-1}	60 s^{-1}	100 s^{-1}
Nominal platen speed	17 $\mu\text{m/s}$	16 mm/s	2.4 m/s	4.8 m/s	8.0 m/s
Actual test speed	17 $\mu\text{m/s}$	16 mm/s	2.6-2.0 m/s	5.0-4.0 m/s	8.3-5.5 m/s
Number of specimens	6	6	4	5	7

The specimens of the low rate tests (2.1×10^{-4} and 0.2 s^{-1}) rested freely on the lower steel platen while the top steel platen descended on the specimen at the designated speed. The force measurement was recorded directly from the load cell attached to the top platen and the displacement was obtained from the moving crosshead. In contrast, the Instron 8800 VHS high rate testing machine has a fixed upper platen with an attached load cell. The bottom platen was then driven upwards using a hydraulic ram to crush the specimen in between. The specimen was fixed on the bottom platen with a small amount of adhesive to prevent it from detaching during its acceleration to the desired test speed. The crush stroke was nominally set to 40 mm. The platen then progressed approximately another 20 mm for the hydraulic system to bring the piston and attached platen to a complete stop at the end of the test. This deceleration phase significantly contributed to the variation in the actual test speed.

3. Results and analysis

3.1. Comparison of damage modes

The typical crush progression of a low rate test is shown in Figure 2. The eight tulip peaks provided trigger points for damage initiation. The sharp tips of the tulip triggers were easily damaged by the loading surface, which created points of initiation where the damage then spread progressively throughout the entire structure. This process maximised the amount of material damaged and hence the total energy absorption whilst preventing high peak load at the onset of crushing. Significant delamination occurred throughout the test. The outer-most ply tended to separate from the bulk of the tube, with delamination advancing a significant distance past the crush front. This ply splayed outwards due to the confines of the platen and the inner plies, which also prompts splitting of the ply parallel to the fibre direction. The lengths of these splits are closely related to the amount of delamination experienced by the ply. As splits form in these outer 0 degree plies, debris falls through the created gaps as evident in Figure 2(d) (circled). The splits

form due to the relatively low matrix strength. Once splitting has occurred, the ply bends and avoids further intralaminar damage caused by subsequent loading.

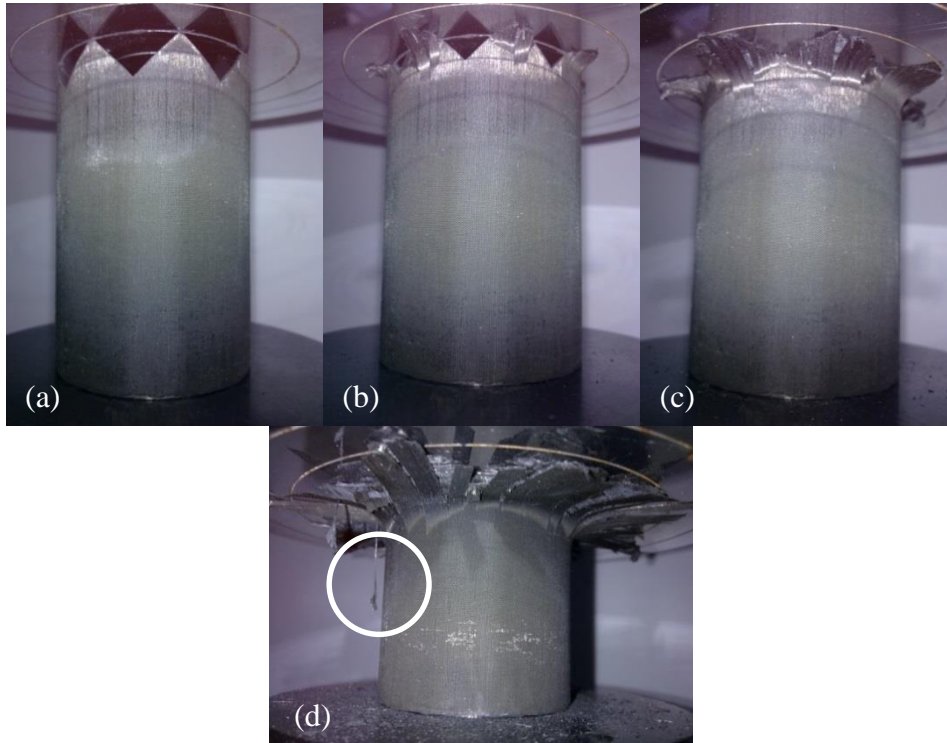


Figure 2: Typical specimen damage progression when crushed at $2.1 \times 10^{-4} \text{ s}^{-1}$: (a) initial, (b) consumption of trigger region, (c) transition to bulk crushing and (d) steady-state crushing of bulk tube.

The crush progression of a typical high rate specimen is shown in Figure 3. The progression is very similar to that of the low speed tests where damage progressively spread from the trigger tips as the crush stroke increased. The outer-most ply delaminated from the main structure and split along the fibre direction to form petals, which remained relatively intact.

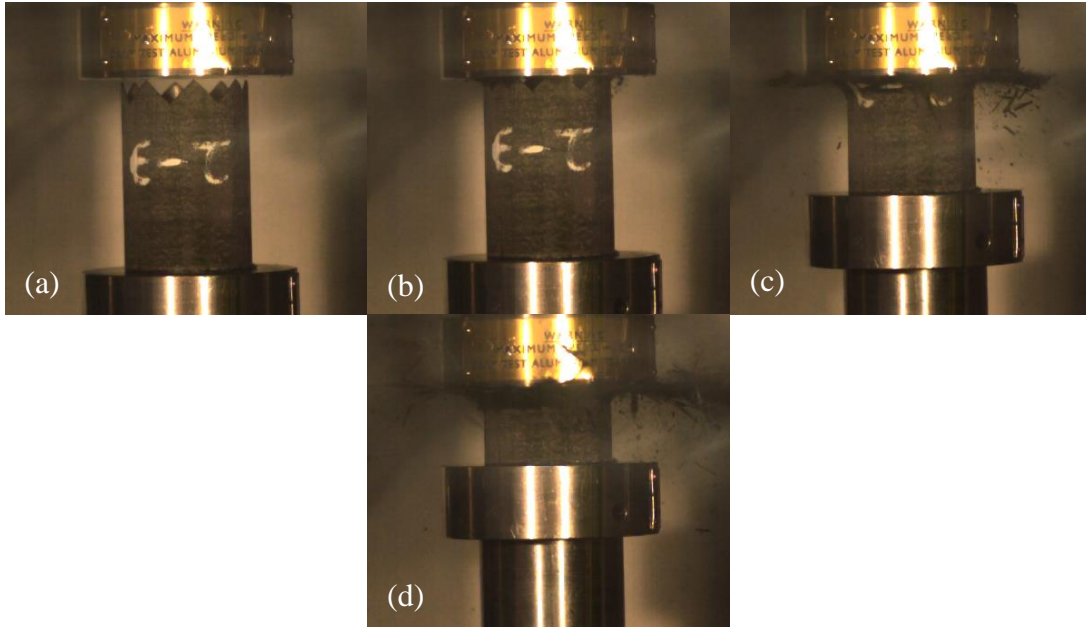


Figure 3: Typical specimen damage progression when crushed at 60 s^{-1} : (a) initial, (b) consumption of trigger region, (c) transition to bulk crushing and (d) steady-state crushing of bulk tube.

A visual inspection of the damage identified in the post-test examinations of typical low and high rate test specimens is presented in Figure 4. In both cases, the inner-most ply exhibited similar behaviour as the outer-most ply. The split segments of the outer plies remained relatively intact in contrast to the internal plies (point A in Figure 4). These outer plies segments formed a funnel-like structure, which trapped debris formed from the disintegration of the internal plies. Due to the confining effects of the outer plies, the internal plies underwent substantial damage that generated a large amount of loosely held debris, regardless of whether they were of 0 or 90 degree orientation. The damage sustained by the internal plies was also consistent between low and high rate tests. The fibres in the 0 degree plies tended to break into smaller pieces due to the confining effects of neighbouring plies. The resulting fragments were typically short and wide (point B in Figure 4). The fibres in the 90 degree plies, which originally ran the circumference of the tube, tended to split into 3 to 4 segments and form thin tendril-like pieces (point C in Figure 4). The similarity in the damage progression and the appearance of the resulting debris in the different tests suggest the damage mechanisms are strain rate independent.

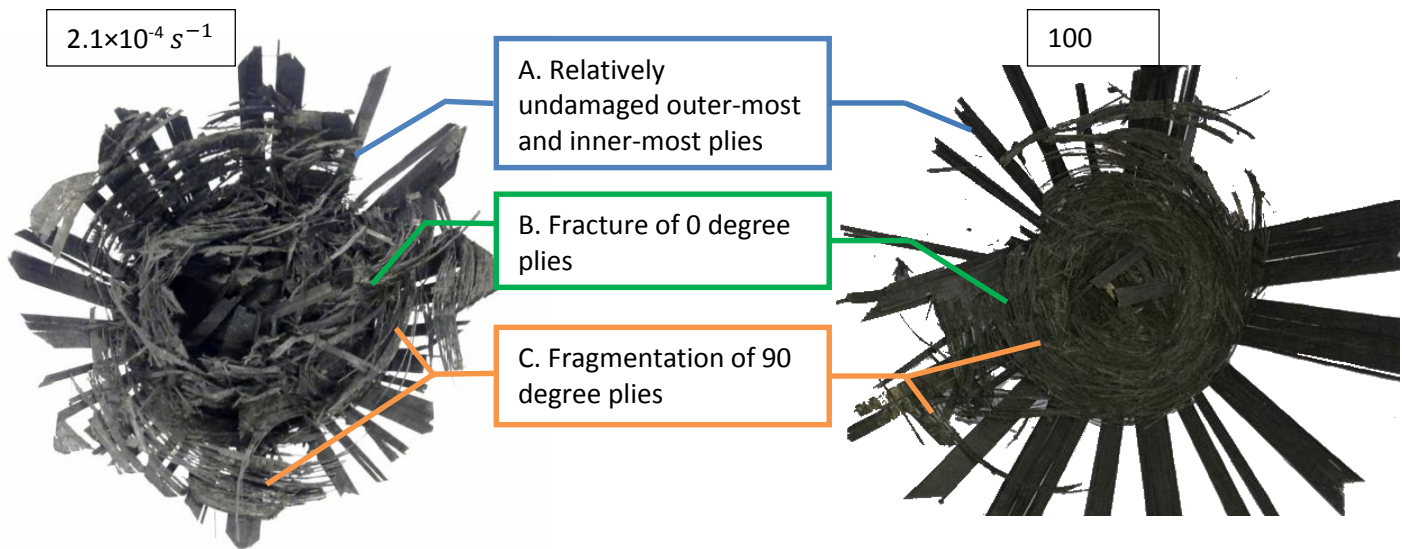


Figure 4: Comparison of damage distribution between specimens tested at different strain rates.

The major difference between the high and low rate specimens is the distribution of debris at the end of the test. The debris in the low rate test largely remained where it was formed whereas the debris in the high rate tests generally moved towards the centre. This was caused by the outer ply extending past the edge of the platen on the hydraulic testing machine. As the platen advanced, the debris would be scraped from the flexible outer-most ply by the edge of the platen towards the centre of the specimen. However, this did not translate to differences in the force response. It was likely that the debris formation process accounted for most of the energy absorption and subsequent motion of the debris was of little significance.

The appearance of the debris shed light into the underlying damage mechanism. In the 0 degree plies, as illustrated in Figure 5(a), the main form of damage was brittle fibre breakage accompanied by matrix shearing. The majority of loading was borne by the fibres. The longitudinal loading induced hoop stresses in the ply, which were resisted by the matrix between the fibres. These stresses caused failure in the matrix, which destabilised the fibre bundles, making them more susceptible to breakage via bending or shearing.

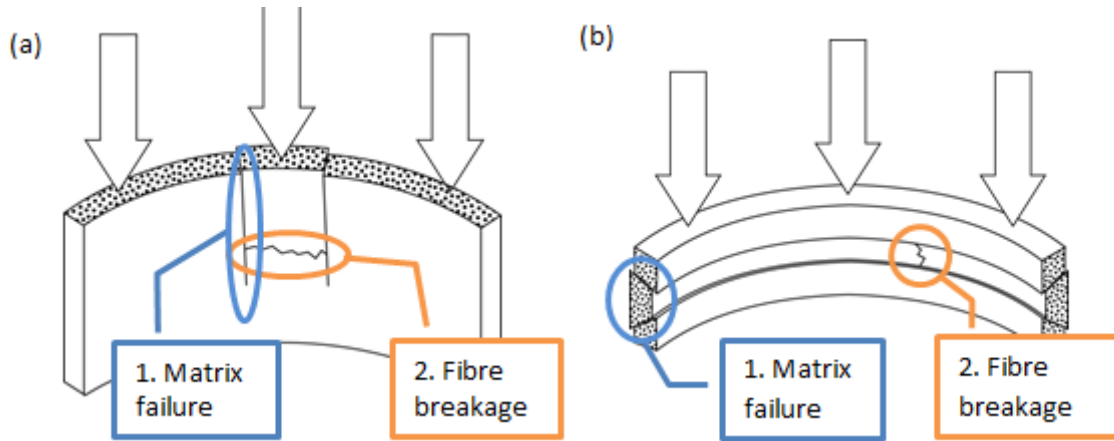
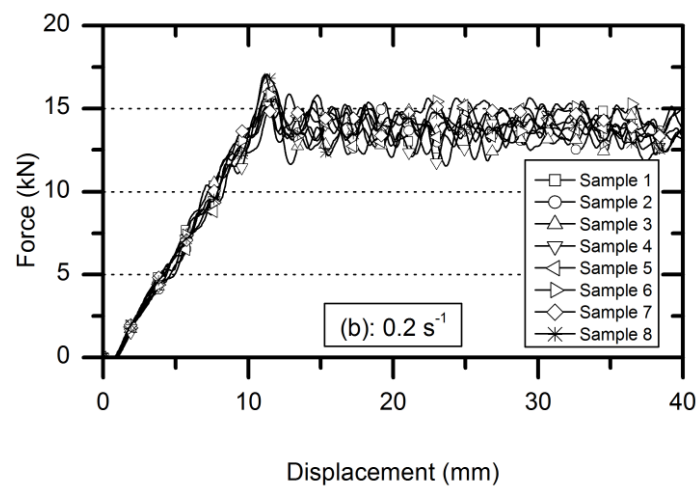
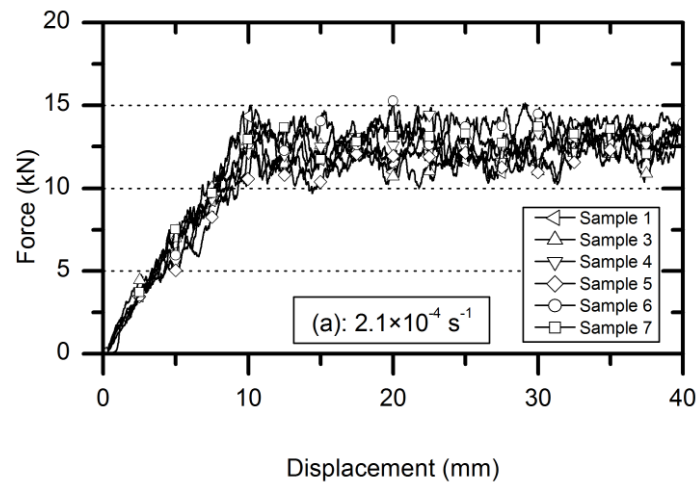


Figure 5: Damage mechanism in the (a) 0 degree plies and (b) 90 degree plies.

In comparison, the damage mode of the 90 degree plies was predominantly shear matrix cracking parallel to the fibre followed by limited breakage of the fibres, as illustrated in Figure 5(b). The shear cracking was triggered by compressive loading transverse to the ply. After the formation of these rings in the presence of adjacent material, further loading tended to cause a change in the circumference of these rings, inducing fibre breakage due to hoop stresses.

3.2. Comparison of force responses

Figure 6 shows the force-displacement response recorded for each set of specimens tested at different strain rates. Due to the controlled crushing of the tulip triggers, the first 10 mm of the stroke saw a smooth, linear increase in the reaction force. This is followed by a transition to steady-state crushing of the bulk cylinder where a slight peak was observed. Once the steady-state crushing was established at approximately 15 mm of stroke, the force level remained constant until the end of the test. All response curves took on a similar form. The increasing test rate increased high frequency noise but did not change the shape of the response curve.



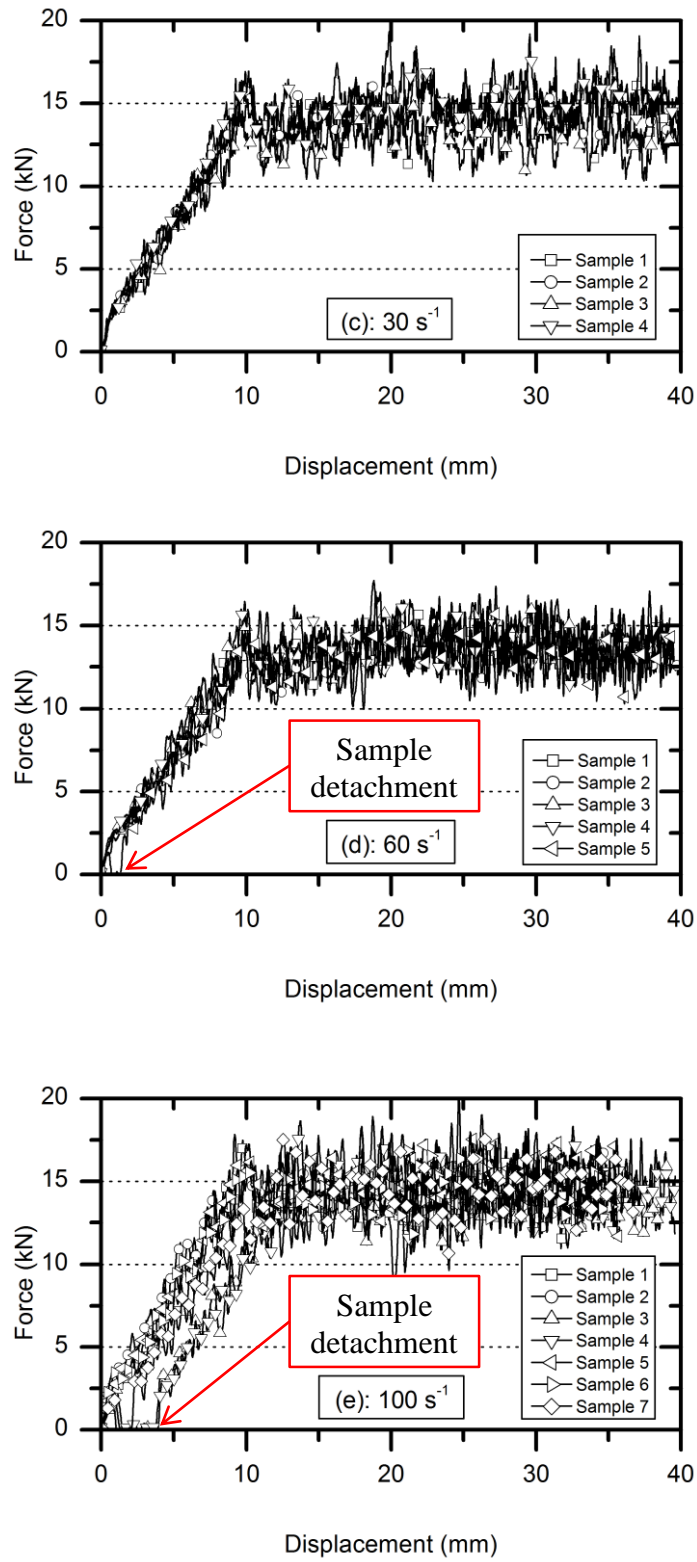


Figure 6: Force response of specimens loaded at (a) 2.1×10^{-4} , (b) 0.2, (c) 30, (d) 60 and (e) 100 s^{-1} , with initial specimen detachment evident in (d) and (e).

Some oscillatory noise can be observed in the force response, particularly in the plateau phase where steady-state crushing occurs. The magnitude of this vibration increased with increasing test rate. The adhesive used to hold the specimen to the platen, as it accelerated to the desired speed, was not sufficiently strong in some cases, particularly at the higher rates. This caused detachment of these specimens, allowing them to hit the other platen prematurely. This detachment caused the gap in the initial stages of the measured force-displacement curve (Figure 6(d) and (e)). Removing this gap yielded a force response that was indistinguishable from the force response of specimens that did not experience detachment, hence these results were retained.

3.3. Comparison of SEA, peak forces, steady-state forces and crush efficiencies at different strain rates

The SEA , F_{peak} , F_{SS} and CE were calculated for each test. Figure 7 shows the averaged SEA for each strain rate along with its standard deviation. The averaged F_{peak} and F_{SS} are plotted in Figure 8 for each strain rate. The progression of CE is plotted in Figure 9.

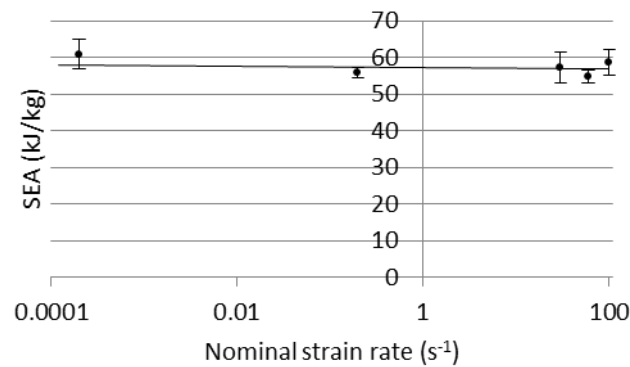


Figure 7: Averaged SEA at different strain rates.

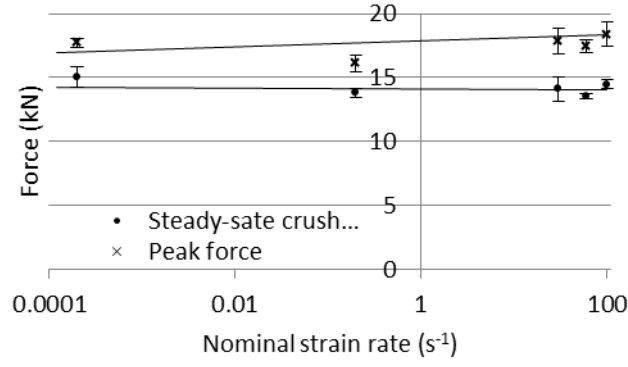


Figure 8: Averaged peak and steady-state crush load at different strain rates.

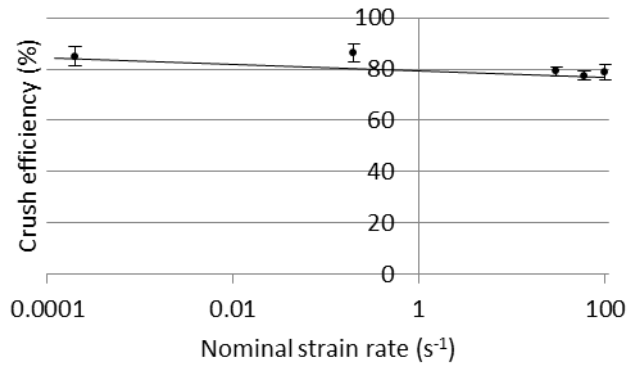


Figure 9: Averaged crush efficiency at different strain rates.

Linear regression was performed to measure the strain rate dependence and the results are shown in Table 2.

Table 2: Results of linear regression performed on calculated metrics

	SEA	F_{peak}	F_{SS}	CE
Regression gradient	-0.0092	0.0151	-0.0008	-0.0008
Std. error in gradient	0.0317	0.0041	0.0034	0.0002
Change over $100 s^{-1}$	2.6%	9.0%	0.6%	8.9%

Both SEA and F_{SS} have small regression gradients and the standard error of these gradients was greater than the magnitude of the gradient. From this, it can be concluded that both SEA and F_{SS} are independent of strain rate. The peak force is expected to occur during transition from triggering to steady-state crushing. However, the peak force usually occurred during the steady-

state crush region in the high rate tests. This suggests that the increasing oscillatory noise is the cause of the increase in observed F_{peak} . The average amplitude of these high frequency oscillations increased from 4.7 kN for the $2 \times 10^{-4} \text{ s}^{-1}$ test to 8.1 kN for the 100 s^{-1} test, which corresponded well to the apparent increase in F_{peak} . Therefore, the increase in F_{peak} with strain rate is likely a result of the vibrations in experimental testing rather than a change in the intrinsic material property. The crush efficiency falls into two distinct groups. The specimens tested on a screw-driven testing machine have higher crush efficiencies than those tested on the hydraulic testing machine. Again, the difference can be attributed to the testing method rather than changes in the underlying material properties.

4. Discussion

4.1. Data reliability

Composite crush structures exhibit complex damage mechanisms, which lead to variability in experimental results. To ensure the validity of the results of the present study, a sample size of between 4 to 7 was used so the scatter in the results of each test configuration can be quantified. In contrast, many authors [7, 8, 18, 24] tested only one specimen per test configuration. The observed damage modes in the present study were also compared against the literature.

The specimens in the present study produced reliable results under the reported testing methods. Pre-test measurement confirmed the consistency in the geometry of specimens. The responses observed between different samples within a batch for each testing condition was also very consistent in terms of the force response as well as the damage pattern. Two typical specimens in the low rate test shown in Figure 10, demonstrate their similarity. Post-test inspections showed that specimens tested under the same conditions were largely indistinguishable from each other and exhibited the same damage modes.



Figure 10: Typical specimens from the $2.1 \times 10^{-4} \text{ s}^{-1}$ test.

Correspondingly, the force response data results also exhibited very little scatter. The value of one standard deviation for all measured metrics under every tested strain rate was smaller than 8%.

The small scatter observed in both the qualitative and the quantitative measurements indicate the reliability of these results.

The damage observed in this study shares many similarities with results in the literature. Huang et al. [26] observed that unconstrained 0 degree plies sustained little damage except splitting along the fibre direction, while the 90 degree plies formed thin strands between the sandwiching 0 degree plies in quasi-static crushing of chamfered cylinders. The higher resin volume fraction in this material caused it to be less fragmented overall in comparison with the present study. The splaying of the outer ply observed was consistent with observations reported by Jackson et al. [17]. Furthermore, the overall size of individual fragments was also similar. In contrast to the loosely held fragments in the present study, the woven nature of Jackson et al.'s [17] specimen led to the fragments being held together.

4.2. *Strain rate independence*

The experimental data shows that the material is insensitive to loading rate for intermediate strain rates of up to 100 s^{-1} . Both low and high rate tests yielded specimens with the same damage modes. The force responses also took on the same form and magnitude. The SEA, peak force, steady-state force and crush efficiency did not vary significantly with increasing strain rates.

Berthe et al. [29] conducted high rate coupon testing on T700GC/M21 material which showed the shear modulus increased by 40% when the loading rate was increased from 10^{-3} s^{-1} to 88 s^{-1} but no discernible change in the transverse modulus. This suggests that the ply configuration is the cause of the apparent rate independence observed in this study.

While the literature is in disagreement over the effect of strain rate on the material response of composite materials [30], there have been suggestions that rate dependent phenomenon is driven by the polymer matrix phase of the composite laminate [31]. Koerber et al. [32] showed rate dependence in the matrix-dominated material properties beyond $200 \text{ } \epsilon/\text{s}$ and rate independence in fibre-dominated longitudinal compressive modulus [33] for a carbon-epoxy material system similar to the one used in this study. The observed rate insensitivity in the present study can be accounted for if rate effects are driven by the matrix material. T700/M21 prepreg has a relatively high fibre volume fraction due to its unidirectional nature. Hence the overall properties of the $[0/90/0/90]_s$ laminate are dominated by the fibre properties, minimising rate dependent effects on the averaged properties of the laminate. Furthermore, the transverse and shear stiffness, strengths as well as the critical energy release rates are all significantly lower than that in the longitudinal stiffness, strength and critical energy release rate. The fibre-dominated critical energy release rates are 108 N/mm and 58.4 N/mm in tension and compression, respectively, in contrast to the 0.331 N/mm and 1.1 N/mm for transverse tension and compression as well as 0.443 N/mm in shear [34]. As the laminate was composed of equal amounts of 0 and 90 degree plies, the bulk of the energy absorption will be attributed to the fibre-dominated failure mode for a given volume of laminate crushed. Again, a change in matrix-dominated energy release rate will have a relatively small effect on the overall energy absorption capability of the laminate.

4.3. *Implications for analysis of energy absorber design*

This result potentially has substantial impact on the design of composite energy absorbers. For materials configured like the T700/M21 cross ply laminate, the fibre-dominated nature of the material response allows strain rate dependence to be neglected. Jacob et al. [27] has identified that an increase in the amount of fibre fracture leads to increased energy absorption. As a result, optimising the crush performance of a structure in the presence of a weight constraint would lead to the fibre-dominated damage mechanism to be the dominant mode of energy absorption. The present results suggest that for these structures, strain rate independence can be assumed.

By discounting strain rate effect, the analysis of energy absorbing structures can be significantly simplified. When rate independence can be assumed, dynamic material properties, which are typically difficult to obtain and require specialised equipment such as the split Hopkinson bar or a hydraulic press, will not be needed. The analysis can also be conducted with rate independent constitutive equations, reducing the complexity and required computational resources of the analysis process. Time scaling, which is a common method to speed up numerical solutions, can be used when the material response is rate independent.

5. Conclusions

The response of a tulip-triggered cylindrical energy absorbing structure undergoing crushing at increasing strain rates was investigated. The damage mechanisms have been identified from post-test inspections. The recorded force response was used to calculate the specific energy absorption (SEA), peak and steady-state forces as well as the crush efficiency.

A high level of consistency was achieved in the present study, indicating good reliability of the results. The observed fracture and damage distribution were indistinguishable between samples tested under the same conditions. Similarly, the standard deviations for the calculated metrics were small. The identified damage modes also correspond well with the literature.

Results indicate that the tested material is strain rate independent up to a strain rate of 100 s^{-1} .

Damage modes and responses were found to be invariant with increasing loading rate experienced by the specimen. Analysis of the specific energy absorption and the steady-state crush force show them to be independent of applied strain rate. Changes in the peak force and the crush efficiency can be attributed to the experimental method rather than a change in the material response. This rate independence is likely the result of a combination of the high fibre content of the laminate and the high fibre-dominated fracture energy release rates in comparison to those of matrix-dominated fracture in the material system used in this study.

For material configurations that are similarly fibre-dominated, the strain rate dependence can be discounted. This speeds up the analysis process of composite energy absorbing structures by reducing the amount of material property data required for input, allowing the analysis to be performed with quasi-static assumptions and allowing for the use of time scaling in numerical models.

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