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Efficient Symmetry-Based Decomposition for Meshing Quasi-Axisymmetric Assemblies


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ABSTRACT

This paper describes a decomposition strategy for hex meshing CAD assembly models consisting of quasi-axisymmetric components. A symmetry-based decomposition technique is used to isolate portions of model boundaries based upon their cyclic symmetry attributes. The calculation of assembly interfaces enables symmetry attributes to be passed between adjacent components with the overall ambition of achieving a decomposition suitable for conformal hex mesh generation of assemblies with quasi-axisymmetric components. The decomposition facilitates a multi-sweep approach to hex meshing where axisymmetric and transition portions of the boundary are sweepable in the circumferential and radial directions respectively. The symmetry-based decomposition is reduced to its equivalent meshable representation where axisymmetric regions are represented by their 2D profile, reflective symmetry is exploited for transition regions and cyclic symmetric regions are defined by a master cyclic segment and a series of transformations. This equivalent meshable representation massively reduces the meshing burden for hex mesh generation.

**Keywords:** Cyclic symmetry, axisymmetric idealization, 3D segmentation, non-manifold modelling, CAD assembly, geometric reasoning, hex meshing, simulation intent.
1 INTRODUCTION

Significant advances in computational hardware and engineering analysis tools have seen a dramatic increase in the size and complexity of the models and physics being analyzed. However, resources are still stretched to their limits when dealing with large CAD assembly models (e.g. whole-engine aero models consisting of thousands of components) and more advanced analyses (e.g. non-linear fan blade-off analyses). In many analyses symmetry properties of components, which include geometry, loading, boundary conditions and material properties, are often exploited by the analyst to reduce simulation run-time by reducing the numbers of degrees of freedom (DOF) in an idealized analysis model. For example, it is commonplace for axial, reflective, cyclic and repetitive symmetry to be exploited for analysis model reduction, provided both geometry and loading exhibit the specified symmetry properties, e.g. a standard axisymmetric analysis model assumes geometry, loads and boundary conditions do not vary in the circumferential direction.

Whilst symmetry idealization of CAD components is a well-established pre-processing tool for analysis model generation, it has not been fully exploited in automatic mesh generation workflows. Mesh generation can often be a major bottleneck in analysis processes, especially for the creation of hexahedral (hex) meshes [16]. In the absence of automatic hex meshing algorithms, the ability to identify symmetry properties enables the partitioning of a complex model into simpler, repeatable sub-domains. For example, detecting and isolating cyclic portions of the model boundary enables one meshed instance to be repeated for all sectors. This reduces the manual meshing burden required to generate hex meshes as only portions of the model need to be meshed, and the full mesh can be generated by applying the appropriate symmetry operators used to partition the model.

Another level of meshing complexity is introduced when dealing with assembly models where there is often a requirement to have conformal meshes between adjacent components. Therefore, assembly configurations require any symmetry-based component decompositions to have compatible interface definitions between adjacent components. In order to achieve this, any symmetry properties must be transferred between adjacent components. This leads to a requirement to define how symmetry properties, and perhaps combinations of symmetry properties, are transferred within assembly configurations. In this work an assembly refers to the Digital Mock-Up (DMU) which consists of the set of geometric components that are spatially positioned within the assembly reference frame without having any explicit link between them. The proposed approach still works on assemblies within which assembly constraints are defined.

In this work the aim is to detect and utilize the symmetry properties of CAD components, in combination with assembly interface information, to decompose quasi-axisymmetric and cyclic-symmetric assembly models into axisymmetric and cyclic-symmetric sub-regions and asymmetric residuals. Tracking assembly interfaces, through symmetry definition changes or after partitioning, enables their symmetry properties to be updated in order to achieve a conformal mesh. The symmetry-based decomposition can be used to extract an equivalent meshable representation, which is a significantly simplified and idealized decomposition. This decomposition is then utilized for hex mesh generation using a multi-sweeping approach for the different regions in the decomposition.

2 BACKGROUND RESEARCH

Incremental approaches are often employed to help automate and reduce the complexity and manual effort associated with the hex-meshing of standalone CAD components. These approaches focus on extracting portions of the geometric model that exhibit certain geometric characteristics which allow the resulting sub-regions to be meshed with robust hex meshing algorithms, such as sweeping and mapping. Robinson [20] and Sun [23] both focus on extracting thin-sheet portions of the geometry (with large lateral dimensions in comparison to the region thickness). Once isolated these regions can be hex-meshed by sweeping a quad mesh through the thickness. Makem [11] and Sun [22] extract long-slender regions (with the dimension along the length of the region much greater than its cross-sectional dimensions). These regions can then be meshed by sweeping a quad mesh of the cross-section along the length of the region. These approaches, amongst others [21], [2], [10], [30], are effectively attempting to generate a sub-division that enables the component to be hex meshed by multi-sweeping [21]. These methods tend to focus on developing geometric reasoning algorithms for certain classes of geometry,
such as thin-walled components. The work in this paper focuses on quasi-axisymmetric components and assemblies, where axial and cyclic symmetry properties are dominant.

Symmetry detection has been widely studied in the computer graphics community and can be classified in terms of global or partial symmetry [25] and exact or approximate symmetry [9]. Numerous methods have focused on finding symmetries in mesh models [13] and B-Rep CAD models [9]. Amongst other applications symmetry detection has been developed for assembly planning [7], detecting design intent [12] and for restructuring feature trees [6].

Limited research efforts have been focused on the use of symmetry properties for automatic mesh generation. Suresh [24] detects global symmetry in 2D cyclic sketches and linearly 3D swept solids. Tautges [26] uses a lattice structure to exploit symmetry and create meshes for nuclear reactor geometry assemblies. These approaches are limited to geometries with simple profiles and obvious sweep directions. This highlights the need for symmetry-based decomposition strategies to extract partial symmetry properties from within complex 3D geometries that exhibit different symmetry properties and can include non-symmetric properties. Cao [3] proposed an approach to use a symmetry-constraint local Delaunay refinement for symmetric 2D or 3D sector meshes. The proposed approach is limited to fully cyclic components and only focuses on tetrahedral mesh generation.

Automatic symmetry detection has often been focused on single components within assemblies without proper consideration of assembly configurations [9]. Vilmart [28] describes the use of a knowledge-based approach combined with the symmetry analysis from Li [9] and geometric interface information to extract repetitive patterns of sets of components within assembly configurations. This allows components to be grouped into their respective families and would be a useful a-priori analysis that could instruct the decomposition method in this work as to which component instances are repeated and only need to be meshed once. Other component instance meshes could be derived from the master component mesh. In this work the aim is to use calculated geometric interface information in the assembly to help propagate symmetry properties between components to facilitate conformal hex meshing of assemblies.

Non-manifold representations have been used to aid the mesh generation of assemblies, where non-manifold topological entities, normally faces, between interacting components provide suitable interfaces for conformal meshing [29]. Qian [18] describes the use of an octree-based approach for generating unstructured hexahedral meshes for manifold domains and non-manifold assembly representations. Non-manifold features, such as the interfaces between components in an assembly, are detected and utilized to modify the base mesh using suitable refinement techniques. However, a certain degree of element distortion remains at the interfaces. Quadros [19] introduced LayTracks 3D, an approach to generate hex-dominant meshes of solids and assembly models. Quadros recognizes the increased complexity when meshing assemblies rather than single components. Assembly imprints are respected and propagated to the medial object (a skeletal representation of the 3D domain) to help generate a conformal mesh. In this paper interfaces and their symmetry properties are combined to instruct and guide a decomposition for hex meshing. Modifications to interface characteristics are tracked and used to enhance the decomposition in order to achieve a conformal mesh.

An alternative to using conformal meshing is to use gluing methods (mathematical constraints between nodes) to join non-conformal meshes [6]. However, gluing approaches can significantly increase analysis solution times and are also unsuitable for certain analyses. Advantages of the gluing approach include the ability to reduce mesh propagation in assemblies and the parallelization of the meshing task. In addition, non-conformal meshes can exist between adjacent components, including facilitating the desired presence of gaps and overlaps. Clark [3] and White [29] also look at addressing assembly gaps and overlaps to obtain a conformal mesh by using tolerant imprinting techniques. In this work only coincident interfaces are considered, with tolerant interfaces left for future research.

3 SYMMETRY PROPERTIES OF COMPONENTS

Symmetrical properties of geometries include axial, cyclic (or rotational), reflective and translational symmetries, or combinations thereof. Quasi-axisymmetric components and assemblies are defined as having a major axis about which most of the geometry is revolved, and large portions of the geometry are axisymmetric. These models can also contain certain non-axisymmetric features. Therefore, in this paper the focus is on the identification and processing of the axisymmetric and cyclic symmetric portions of the geometry that are prevalent in quasi-axisymmetric components.
Symmetry around a central point in 2D, or an axis line in 3D, is referred to as cyclic symmetry and is denoted by \( C_n \), where \( n \) is the number of repetitions in the cyclic pattern. This \( n \)-fold cyclic symmetry property helps describe a set of rotation transformations \( R_{C_n} \) where the angle or rotation is defined by \( \alpha = 2\pi i/n \) where \( n \geq 2 \) and \( i \in [1, n - 1] \). Axisymmetric properties are considered a special case of cyclic symmetry where \( n = \infty \) and axis-symmetry is denoted by \( C_\infty \).

![Fig. 1: Global symmetry-based decomposition: (a) B-Rep CAD model exhibiting \( C_5 \) cyclic symmetry, (b) cyclic sector, (c) sector decomposed into sweepable bodies for multi-sweeping with arrows representing sweep direction for each sub-region, (d) sector mesh, (e) fully patterned mesh.](image)

Global symmetry properties of a component describe a transformation that maps the entire object to itself. Axisymmetric and fully cyclic components exhibit global symmetry around a global axis of rotation that passes through the component centroid. A component can have multiple reflective symmetry properties through multiple global reflective planes that must pass through its centroid. Partial symmetries refer to multiple combinations of symmetric and non-symmetric properties existing in a single component. These symmetries are represented through the decomposition of a component into sub-regions in order to isolate the distinct symmetry definitions. Each distinct partial symmetry transformation is around a global axis or plane of the original component. Fig. 1 illustrates an example exhibiting global cyclic symmetry, \( C_5 \) (i.e. there are five instances). Once the cyclic sector has been extracted, further manual decomposition is carried out to facilitate a multi-sweep meshing approach to generate the sector mesh. Cyclic, reflection and translation symmetry transformations are Euclidean maps as original lengths and angles remain invariant under transformation. They can be applied to the meshes generated on components exhibiting these symmetry characteristics. This enables the full component mesh in Fig. 1 (e) to be generated using the \( R_{C_5} \) transformations applied to the cyclic sector mesh shown in Fig. 1 (d).

![Fig. 2: Partial symmetry-based decomposition using Fig. 1 (a) as input model: (a) Isolation of symmetry regions, axisymmetric (\( C_\infty \)) in green, cyclic (\( C_5 \)) in red, (b) equivalent meshable representation, (c) axisymmetric mesh generation, (d) cyclic mesh generation, (e) full component mesh.](image)

Whilst global symmetry can be exploited for simplifying FEA models, it is often desirable to extract local or partial symmetry properties to aid with hex mesh generation, especially for complex CAD

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models where global symmetry often doesn’t exist. Local symmetries are defined using local axes and planes within a model. In this work we limit the detection to global and partial symmetries around a global axis of rotation (the axis of rotation is user-specified). Local symmetry detection will be a subject of future work. Fig. 2 shows decomposition and meshing of the component in Fig. 1 using the process described in [1] to extract the partial symmetry properties and in-effect generating similar decompositions as required for the multi-sweep in Fig. 1 (c). This decomposition isolates axisymmetric and cyclic symmetric regions in the component automatically, Fig. 2 (a), and generates an equivalent meshable representation suitable for hex meshing, Fig. 2 (b), where axisymmetric regions are represented by their 2D axisymmetric profile, Fig. 2 (c). This equivalent meshable representation consists of the unique primitives, alongside their associated transformation matrices, required to generate the full component mesh, Fig. 2 (e).

4 SYMMETRY-BASED DECOMPOSITION FOR HEX MESHING

Previous work [1] by the same authors describes a symmetry detection and decomposition approach for B-Rep CAD models, depicted in Fig. 3. The main hypothesis is to classify faces as axisymmetric, cyclic symmetric, pseudo-axisymmetric and non-axisymmetric and use these properties to decompose the original model into meshable subsets, including axisymmetric and repeatable cyclic regions. In this work the global symmetry axis is supplied as input by the user. Axisymmetric and pseudo-axisymmetric faces are first identified. Axisymmetric faces have axisymmetric surface and boundary definitions, e.g. a cylinder with two peripheral loops. Pseudo-axisymmetric faces have axisymmetric surface definitions but non-axisymmetric boundaries, e.g. cylindrical faces with inner loops or non-axisymmetric peripheral edges [1]. Quasi-axisymmetric properties relate to the global properties of a body, whereas pseudo-axisymmetric properties relate to the local properties of faces.

Fig. 3: Symmetry-based decomposition: (a) CAD assembly, (b) outer casing identified as axisymmetric (green), (c) identification of cyclic faces and groups (red faces are the master group and grey the repeated), (d) decomposition of hub into axisymmetric (green), cyclic (red is cyclic and grey is repeated).

Cyclic faces are geometrically and topologically congruent and are repeated around a common axis of rotation. Cyclic symmetry detection outputs repeated sets of cyclic faces where each face in the set is congruent with all other repeated entities. These sets of faces are then grouped into cyclic group patterns, denoted as $G_n$ with $n$-fold symmetry consisting of a combination of cyclic faces defined within that pattern. A single component can consist of $m$ cyclic group patterns. As for face sets, each cyclic group is congruent to all others in the group set. Any remaining faces are classified as non-axisymmetric.

Once faces have been classified into their applicable symmetry group, symmetric portions of the boundary are isolated as described in [1]. Fig. 3 (b) shows that for the assembly in Fig. 3 (a) which consists of two components shown in grey and purple, the outer casing is identified as axisymmetric since it is bounded by all axisymmetric faces. Boundary information of axisymmetric faces and suitable projections from pseudo-axisymmetric faces are used to isolate axisymmetric regions. After isolating all of the axisymmetric portions of the model boundary, Fig. 3 (d) green subsets, cyclic portions of the boundary are isolated, Fig. 3 (c) red. Prior to decomposition all faces in the cyclic boundary portion must be cyclic, axisymmetric or pseudo-axisymmetric. After decomposition all faces become cyclic with
attached transformations to define any repeated definitions. The limitation with the decomposition in Fig. 3 and the work in [1] is that there is no consideration of the assembly configuration and relationships during the decomposition. This yields an independent decomposition for each component in the assembly. Therefore, mesh conformity at component interfaces is not assured.

4.1 Interface guided decomposition

Whilst a valid hex mesh can be generated for each of the sub-domains in the decomposition shown in Fig. 3 (b) and (c), each cell will be meshed without considering its adjacent cells and so connection elements are required to couple the incompatible meshes of the inner and outer structures. In order to achieve a fully contiguous hex mesh the interfaces between components need to be identified. To achieve this Fig. 4 shows the workflow for the overall assembly decomposition approach.

Fig. 4: Workflow of assembly decomposition approach.
Compatible interface definitions require the imprinting of adjacent component boundaries on one another. Non-manifold Boolean union operations are used to combine components into a single non-manifold assembly with multiple connected sub-regions from which the desired non-manifold interfaces are extracted, Fig. 5 (a). Interface definitions between components are identified in a standalone non-manifold application developed using the Parasolid Geometric Modelling Kernel [17]. The interface definitions are calculated on a tagged Parasolid assembly file automatically exported from within Siemens NX [15] and then transferred back within the manifold CAD environment of Siemens NX which is the tool used for the symmetry detection, decomposition and meshing. This is achieved by using the appropriate non-manifold boundary information, extracted from the Parasolid tool, to imprint the equivalent manifold faces in Siemens NX, thus creating exact matching faces either side of the interface, Fig. 5 (b), and a fully connected assembly representation. Since the non-manifold interfaces are propagated to the manifold assembly in NX, the Parasolid model can be discarded once the information has been extracted. These interface definitions are defined after the initial classification of component faces into their respective symmetry groups.

After the imprinting operation has been carried out, the simulation attributes of the assembly interfaces, in this case their symmetry properties, are automatically propagated between components. This approach is different from the work in [28] where interfaces and component symmetries are used to form relationships between collections of components to help identify sub-assembly symmetries. Keskin et al. [8] describe a novel contact meshing approach whereby the hex meshing process is simplified by allowing mismatching meshes at ‘glued’ interfaces. This enabled the generation of all-hex meshes with better quality and a lower total element count but with the drawback of stress and displacement error in the contact regions. Here the symmetry properties are transferred from one component to another to help generate a fully compatible and connected hex mesh at the interfaces. The main assembly interface types that have been considered are based on the symmetry attributes (axisymmetric, pseudo-axisymmetric, and cyclic) of the faces and are classified based on the interaction of these properties. How these interfaces are treated dictates the downstream meshing recipe and the equivalent meshable representation that is created, Section 4.3.

4.1.1 Axisymmetric-axisymmetric interfaces

For contiguous meshing purposes interfaces between axisymmetric bodies can be treated in one of two ways. Either the axisymmetric bodies on both sides of the interface can be merged, or the same mesh density can be assigned to all axisymmetric edges in adjacent bodies. This is all that is required to generate a compatible mesh between axisymmetric regions. However, to generate this mesh, matching axisymmetric faces must be generated either side of the interface. Axisymmetric-axisymmetric interfaces can only interact in three different ways:

- Boundaries of the axisymmetric faces are coincident and therefore the axisymmetric faces are exact matches.
- Axisymmetric faces overlap, i.e. none of boundaries of the axisymmetric faces are coincident. This means three new axisymmetric faces will be generated with one in common to both axisymmetric bodies, and one unique to each component
- One axisymmetric face is engulfed by the other. The engulfed face will be common between both bodies and either one or two new axisymmetric faces will be generated depending whether one of the boundary edges is coincident.

From the possible axisymmetric interactions listed above it is obvious that only axisymmetric faces can be generated from the interaction between two axisymmetric faces.
Fig. 5: Interface guided decomposition: (a) assembly interfaces, (b) imprinted interfaces, (c) new decomposition of outer casing, (d) new symmetry properties of outer casing.

4.1.2 Axisymmetric-cyclic interfaces

For axisymmetric-cyclic interface types the axisymmetric faces will always inherit the properties of their adjacent cyclic faces from another component. As the axisymmetric and cyclic faces share the same axis of rotation they can only interact in two ways:

- The cyclic faces are engulfed by the axisymmetric face.
- The cyclic faces intersect the axisymmetric boundary.

From either scenario above, the axisymmetric face is partitioned into a pseudo-axisymmetric face (bounded by the outer boundary of the cyclic faces and the original axisymmetric edges) and a number of cyclic faces. The number of partitioned cyclic faces is equal to the number of loops in a cyclic face (outer and inner loops) multiplied by the cyclic pattern $C_n$. The cyclic boundaries are then used to decompose the axisymmetric face. Fig. 5 (b) shows the imprinted faces on the outer casing which inherit the $C_{10}$ cyclic symmetry properties of their adjacent cyclic faces. Since there are two loops in each cyclic face (one outer loop and one inner loop) the number of subset cyclic faces is 20 as the cyclic pattern is $C_{10}$. Half of these faces are one-one maps with the original cyclic faces themselves, the other half are new cyclic faces generated from the cyclic inner loops. As a result the cyclic groups and face classifications specified are automatically updated after the imprinting operation.

Using the methodology from [1] the outer casing is then decomposed into the axisymmetric and cyclic regions as shown in Fig. 5 (c). Pseudo-axisymmetric-cyclic interfaces are treated in the same manner as axisymmetric-cyclic interfaces, provided there is no interaction between the cyclic faces and the non-axisymmetric entities bounding the pseudo-axisymmetric face.

4.1.3 Axisymmetric-pseudo-axisymmetric interface

A number of different configurations can exist between interacting axisymmetric and pseudo-axisymmetric faces. These configurations depend on the symmetry classification of the edges bounding the pseudo-axisymmetric face and also the nature of the interaction. Fig. 6 depicts some axisymmetric-pseudo-axisymmetric interface interactions and shows how the symmetry properties of the interface can differ from that of the interacting face where:

- Fig. 6 (a) shows how the pseudo-axisymmetric boundary intersects the axisymmetric faces generating an exact imprint of the original pseudo-axisymmetric face (grey) Fig. 6 (b) and a series of cyclic faces (red). Thus, cyclic faces (red) are generated where cyclic boundary edges of a pseudo-axisymmetric face isolate a subset of an axisymmetric face.
- Another possible scenario is illustrated in Fig. 6 (c) where the pseudo-axisymmetric interface is engulfed by the axisymmetric face. This results in the generation of two pseudo-axisymmetric faces.
- Once boundaries of an axisymmetric face are used to partition a pseudo-axisymmetric face the resulting imprinted faces can be either cyclic, axisymmetric or pseudo-axisymmetric depending on the type of interaction and the symmetry properties of the edges bounding the pseudo-axisymmetric face, Fig. 6 (e-h).
The important point is that not only do the symmetry properties of the face itself dictate the interface type, but also the symmetry properties of the edges bounding those faces, especially the edges bounding pseudo-axisymmetric faces. Any non-axisymmetric portions of the pseudo-axisymmetric face will be reflected in the adjacent component and subsequently isolated as a non-axisymmetric volume. Likewise, any inner loop in a pseudo-axisymmetric interface will generate a new face on the opposing face which will inherit the symmetry properties of the inner edges.

Fig. 6: Axisymmetric-pseudo-axisymmetric interactions: (a) two component assembly with pseudo-axisymmetric (left) axisymmetric (right) interface, (b) imprinted interfaces with cyclic (red) and pseudo-axisymmetric (grey) of (a), (c) pseudo-asymmetric interface reduced, (d) imprinted interface of (c), (e) pseudo-axisymmetric interface increased, (f) imprinted interface of (e), (g) axisymmetric interface reduced, (h) imprinted interface of (g).

4.1.4 Cyclic-cyclic interfaces

Cyclic-cyclic interfaces will generally only occur in assemblies when cyclic faces from interacting components have the same, or similar, cyclic patterns. In addition, the surface definitions of interacting components will be compatible but they may have different face topologies. Variants in cyclic patterns between assembly interfaces are most likely to occur after the decomposition where cyclic patterns from within components propagate to the interface region. This scenario is presented in the next section.

4.2 Tracking assembly interfaces

The previous sub-section demonstrated how interfaces are treated based upon their symmetry properties. This section describes how interfaces must be tracked throughout the decomposition to determine if their symmetry classification and their resulting interaction is altered due to partitioning at the interface. After the imprinting operations the mapping between matching coincident faces is stored, i.e. two manifold faces either side of an interface are paired together. This mapping is tracked after the decomposition phase in order to process interfaces that have been partitioned and whose symmetry definition has been modified.

Fig. 7 shows the tracking and updating of assembly interface definitions for two adjacent components with different cyclic patterns. The original assembly interfaces in Fig. 7 (a) exhibit axisymmetric symmetry. However, once the decomposition has been performed, Fig. 7 (b), it can be seen that the original axisymmetric interfaces have been modified to become two distinct cyclic symmetric definitions, shown in Fig. 7 (d) and (e). In order to achieve a conformal mesh these symmetry changes must be propagated between the interface regions. Fig. 7 (f) and (g) show imprints (highlighted in red)
from one interface onto the other. In effect, this imprint operation can either change or maintain the symmetry definition of these interfaces once again. On one side the $C_{10}$ master interface is partitioned into two cyclic symmetric faces, Fig. 7 (g). This means the master interface maintains its $C_{10}$ cyclic relationships with its slave interfaces enabling the master mesh (red) to be repeated for the slave meshes (grey), Fig. 7 (h).

On the other side of the interface the $C_4$ master interface is partitioned into three different faces, of which two are cyclic. Based upon the partitioning after the imprinting operation the relationship between the $C_4$ master and slave interfaces is modified to a $C_2$ pattern, Fig. 7 (i), with a reflective pattern in the y-axis, Fig. 7 (j). However, in meshing terms, since the two cyclic subset faces in the $C_4$ master interface is a slave interface in the $C_{10}$ interface definitions it is possible the cyclic pattern between $C_4$ master and slave interfaces is maintained and used to generate the patterned mesh, Fig. 7 (k).

(a)  
(b)  
(c)  
(d)  
(e)  
(f)  
(g)  
(h)  
(i)  
(j)  
(k)  

Fig. 7: Tracking interfaces after decomposition: (a) two component assembly, (b) decomposition with different cyclic patterns, (c) front view of interface patterns, (d) modified $C_{10}$ interface, (e) modified $C_4$ interface, (f) $C_{10}$ imprint on $C_4$, (g) $C_4$ imprint on $C_{10}$, (h) cyclic mesh for (g), (i) cyclic pattern modified after imprint, (j) new reflective interface symmetry, (k) cyclic mesh of (f).

4.3 Equivalent meshable representation

Simulation Intent [14] refers to the high-level analysis decisions required to generate fit-for-purpose analysis models. Different Simulation Intent definitions can be utilized on the same design geometry to produce the different analysis models that may be required for different applications. Tab. 1 illustrates how different simulation intent definitions are used to create different decompositions required to
generate fit-for-purpose meshes. An initial decomposition is carried out to isolate the axisymmetric and cyclic regions in the model, first decomposition in case 1 in Tab. 1. This initial decomposition corresponds to the first simulation intent definition of the analyst where a local fine mesh is desired for a detailed stress analysis. Depending on the simulation intent new appropriate decompositions are also produced to satisfy analysis requirements. For example, case 2 in Tab. 1 corresponds to an analysis requirement to reduce the number of DOF in the model for a global assessment of mechanical behavior.

<table>
<thead>
<tr>
<th>Decomposition</th>
<th>Equivalent meshable representation</th>
<th>Equivalent mesh</th>
<th>Full conformal assembly mesh</th>
</tr>
</thead>
<tbody>
<tr>
<td>Case 1</td>
<td><img src="image1.png" alt="Image" /></td>
<td><img src="image2.png" alt="Image" /></td>
<td><img src="image3.png" alt="Image" /></td>
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<tr>
<td>Case 2</td>
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<td><img src="image5.png" alt="Image" /></td>
<td><img src="image6.png" alt="Image" /></td>
</tr>
</tbody>
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Tab. 1: Different decompositions and equivalent mesh representations based on simulation intent

Once the appropriate decompositions are generated the analysis representation is reduced to its equivalent meshable representation. The equivalent meshable representation is the minimum number of cells in the specific assembly decomposition from which the full assembly mesh can be generated by applying the sweep operations or transformation operators defined during the decomposition phase. As an enhancement to the work in [1], here the equivalent meshable representations are automatically derived from the decomposition and the full hex meshes are automatically generated for the different
Simulation Intent definitions and are shown in Tab. 1. It is also demonstrated how different equivalent meshable representations are generated from the same Simulation Intent definition but can generate the same resulting mesh, e.g. the equivalent meshable representations generated for the case 1 decomposition. Thus, two different equivalent meshable representations can be automatically generated to define the same fine mesh. The only difference being in how the axisymmetric regions are treated.

As the first decomposition (case 1 in Tab. 1) is produced in order to yield a fine mesh the small element size from the cyclic regions is propagated through adjacent axisymmetric regions. This is achieved using either of the two following equivalent meshable representations.

The first equivalent meshable representation that can be utilized to achieve this fine mesh is to propagate the cyclic partitions into the axisymmetric regions. This means that the axisymmetric cells in the decomposition are converted to $C_{10}$ cyclic cells, which makes them sweepable in the circumferential direction. The mesh density is propagated across the cyclic-axisymmetric interface and the resulting swept mesh is repeated using the cyclic transformation $R_{C_{10}}$ for all the cyclic subsets in the decomposition.

The second equivalent meshable representation idealizes the axisymmetric regions to their equivalent 2D axisymmetric profile. The 2D idealized representations are then quad meshed and revolved to generate $C_{10} \times N$ hex elements around the circumference, where $N$ is the number of nodes on the cyclic-axisymmetric interface. For both options any cyclic periodic faces (the cyclic partitioning faces) are constrained to have compatible division numbers on equivalent cyclic periodic edges, thus enabling fully contiguous hex meshes to be generated.

As the second decomposition (case 2 Tab. 1) is defined for an analysis where there is a requirement to reduce the number of DOF in the hex mesh it is desirable to transition between smaller mesh densities in the cyclic regions, where typically detail exists, to larger mesh densities in adjacent axisymmetric regions. To achieve this the axisymmetric regions are decomposed further, with an axisymmetric partition, to create transition regions at the axisymmetric-cyclic interfaces, case 2 Tab. 1 blue regions. In transition regions the mesh is swept through the thickness of the region. This is in contrast to axisymmetric regions that are swept along the circumferential direction. Since the transition regions originate from an axisymmetric region they contain $C_r$ reflective planes. Therefore, only half of the transition regions need to be represented in the equivalent meshable representation. A reflection transformation is used to generate the remainder of the mesh.

The more detailed mesh from case 1 above consists of 38,870 element and 75,480 nodes while the more efficient mesh generated in case 2 consists of 12,024 elements and 22,728 nodes. This shows a 70% reduction in the number of elements and nodes between the two models. For case 1 two equivalent representations are used to generate the same final mesh of the assembly. The first representation is a fully solid decomposition requiring the direct generation of 2,719 hex elements. This represents only 7% of the overall mesh, with the remaining 93% of elements being cheaply patterned. The second representation is a mixed-dimensional model consisting of 2D axisymmetric profiles and solid cyclic regions. This representation requires the direct generation of 823 hex elements and 110 quad elements, which is only 2.5% of the overall mesh, where the remaining 97.5% of elements generated through either patterning or revolve operations. The decomposition in case 2 produces a mixed-dimensional model where only 2,168 hex elements and 47 quad elements are directly generated. This represents 18% of the overall mesh for this decomposition as the transition zones consume some of the axisymmetric region and the mesh is much coarser than that of case 1. A benchmark tet meshed model with global 10mm element size (taken as average from finer hex mesh of case 1) consists of 110,000 elements. Both the fine and efficient hex meshes are 65% and 89% less respectively. It is worth mentioning this gain is with only one element in the thickness, and as more elements are utilized in the thickness the gains will become more apparent.

It is worth noting that the aim of this work is to show the automated workflow of the decomposition strategy, how this is represented as the minimal decomposition and how this can be meshed and transformed to generate the complete mesh of the assembly. Validation to date has focused on achieving a fully automated workflow producing a fully connected mesh with no hanging nodes. Therefore, little effort has focused on mesh quality, such as ensuring there are a suitable number of elements through the thickness which can differ for different applications. Such improvements will be the topic of future work where for example multi-block decomposition methods will be used on the axisymmetric profiles and element division numbers will be automatically calculated using integer programming routines whose constraints can be informed by the symmetry properties extracted during the decomposition.

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Fig. 8: Interface guided decomposition for use-case industrial example: (a) Original assembly, (b) section view of original assembly, (c) symmetry-based decomposition of assembly, (d) section view of (c), finally (e) is the equivalent meshable decomposition of (c).

5 RESULTS AND DISCUSSION

Fig. 8 (a) shows the use-case utilized in this research which is a simplified aero-engine model supplied for academic research as part of the CRESCENDO project [5]. The assembly consists of 5 components with a total of 1112 faces, 2381 edges and 1433 vertices. The described approach has been implemented using the C# language and .NET framework APIs in Siemens NX [15] and Parasolid [17]. The automated identification and decomposition, shown in Fig. 8, takes 3 minutes 12s (on 64-Bit Windows machine with
a 3.7GHz Intel Zeon E5-1630 CPU with 32GB RAM). This includes the symmetry-based decomposition, Fig. 8 (c) and (d), and the generation of the equivalent meshable representation, Fig. 8 (e), which comprises the tracing and creation of the axisymmetric 2D profiles. The equivalent meshable representation resulting from the decomposition reduces the complexity of the hex meshing problem where: 60% of the original model volume is now represented as axisymmetric bodies (green); 28% is identified as cyclic (red and grey); 5% as transition regions (cyan) and 4% is non-axisymmetric (yellow). The remaining 3% is comprised of the sweepable / long-slender (blue) and block topology regions. Therefore 60% of the original volume has been reduced to a 2D meshing problem, as the 2D axisymmetric profile are quad meshed and revolved. Transition, sweepable and block regions can all be easily meshed with existing sweep and mapping algorithms. This leaves only the cyclic and non-axisymmetric regions to be dealt with. Since the cyclic region meshes are generated with n repetitions of a master mesh, only 3% of the cyclic subsets need meshed with the residual 25% being repeated. Of this 3% of master cyclic cells, approximately half are identifiable as sweepable (using the topological algorithm in [27]). This means with only the remaining 1.5% cyclic and 4% non-axisymmetric cells there is merely 5.5% of the original assembly that needs to be further processed for hex meshing.

Currently the example in Fig. 8 cannot be automatically meshed due to the further effort required to mesh the un-sweepable cyclic and non-axisymmetric subset cells. However, the example provided in Tab. 1 shows the generation of two distinct symmetry-based decompositions from an original assembly. The different decompositions facilitate the creation of different fit-for-purpose meshes. The decomposition, generation of equivalent meshable representations, generation of the meshing recipe and meshing are all automated for the example in Tab. 1, with each automated process taking <20seconds. It is also shown that two equivalent meshable representations can be used to generate the same mesh. Future work will extend the automated meshing portion of the tool to deal with more complicated decompositions, as shown in Fig. 8, with the ambition of building the technology for extracting thin-sheet [23] and long-slender regions [22] into the current workflow.

Section 4.3 described how interfaces need to be tracked in order to propagate any symmetry modifications between the interface regions. A detailed example is provided where the cyclic pattern is different between adjacent regions after decomposition. Section 4.3 demonstrated how these changes can be passed between components using imprint operations to reflect the symmetry change between interfaces. Another way of addressing this is to push the problem downstream, where correct assignment of edge division numbers can be used to achieve the same result as the imprinting operation. However, this is dependent on the compatibility between mismatching interface symmetries where division number assignment effectively requires the calculation of a minimum angle factor between cyclic partitions in order to achieve a conformal mesh. One drawback of the minimum angle factor approach to division assignment would be, in certain instances, the creation of an overly dense mesh. This is because, at present, exactly the same mesh is utilized for cyclic master and slave representations, where based on interface types different division numbers may need to be assigned to master and slave edges. All of these issues will be addressed in future work where more focus will be on the actual mesh generation.

This focus will include the inclusion of transition regions after decomposition is enhanced to cater for the mismatching symmetry definitions at interfaces. Since these mismatching interfaces will generally produce a refined mesh at the interface one issue will be to assign appropriate ‘cyclic transition regions’ to manage the effect of interface mesh through the rest of the region. This of course depends on the fit-for-purpose analysis model required.

Interface calculations have been performed in this work using non-manifold Boolean unions to extract common faces between assembly components. To date this has focused solely on coincident interfaces with tolerant interfaces (gaps and overlaps) being left for future research. Interfaces are transferred from the non-manifold representation in Parasolid to the manifold one in NX. Tracking interfaces in the manifold environment becomes challenging, especially once they are partitioned during the decomposition. It is the aim of the authors to implement this process using their previous research on Virtual Topology decomposition of volumes [27]. This will also help avoid any tolerance issues encountered during the geometric decomposition and will prove useful when tackling assemblies with gaps and overlaps.
While the work presented in this paper has made a significant contribution to the achieving a symmetry-based decomposition of quasi-axisymmetric assemblies there remains room for improvement. The equivalent meshable representation generates a reduced subset of meshable entities that can be meshed and transformed to generate a mesh of the full assembly. However, this only takes into account symmetries within components and between assembly interfaces. For certain assemblies the equivalent meshable representation could be reduced further by exploiting assembly symmetries, such as those identified in [28], which would enable repetitive components to be represented by a designated master component and associated transformations. This fits nicely into the notion of the equivalent meshable representation which aims at reducing the overall meshing burden / complexity.

Another logical progression of this work is the ability to treat moving components and to update the decomposition and resulting mesh automatically. One interesting aspect of treating movement is how the interactions between symmetries at the interfaces will be processed and how this will constrain the downstream meshing process. For example, for rotating components the decomposition strategy will need to be altered to avoid the creation of sliver entities at the interface. Another consideration when treating moving interfaces is the possibility of mesh inconsistencies at the interface, e.g. mismatching meshes may result when components either side of the interface move or mesh penetration may occur, where nodes on one side of the interface penetrate elements on the other side. In this work the interface between adjacent components is explicit, meaning once common mesh is correctly defined between them by assigning the correct division numbers, these issues can be managed at the interface.

6 CONCLUSIONS
This paper introduces a symmetry-based decomposition approach for quasi-axisymmetric CAD assembly models. Faces in all components are classified based upon their symmetry properties. Next interfaces between components are identified and used to propagate symmetry properties between adjacent components. Those interfaces completely engulfed by their adjacent face transfer their symmetry definition to the adjacent components. Where boundaries of interface regions intersect the resulting imprinted faces inherit symmetry from the partitioning edges. Axisymmetric and cyclic portions are extracted from each component. Interfaces are re-evaluated to determine if their symmetry properties have been modified during the component decomposition. Modified interfaces are utilized to generate an enhanced decomposition consisting of compatible interface suitable for conformal meshing.

Different equivalent meshable representations and therefore different fit-for-purpose hex meshes are automatically defined depending on the simulation intent specified by the analyst. Distinct equivalent meshable representations can be generated from the decomposed geometry, both of which adhere to the same Simulation Intent definition and are used to create the same fit-for-purpose mesh. Also, different Simulation Intent definitions can be used to create considerably different decompositions for the same design geometry. Utilizing the equivalent meshable representations with the appropriate transformations significantly reduces the burden required to mesh the overall assembly, as can be seen in the use-case where less than 6% of the model wasn't instantly hex meshable using this approach.

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